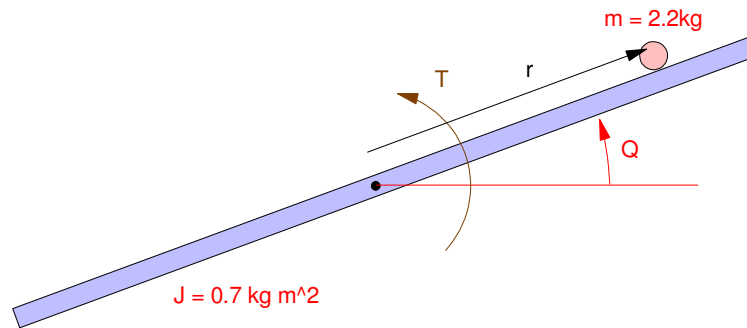


ECE 463/663 - Homework #7

Servo Compensators. Due Monday, March 17th



The dynamics of a Ball and Beam System (homework set #4) with a disturbance are

$$s \begin{bmatrix} r \\ \theta \\ \dot{r} \\ \dot{\theta} \end{bmatrix} = \begin{bmatrix} 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \\ 0 & -7 & 0 & 0 \\ -7.434 & 0 & 0 & 0 \end{bmatrix} \begin{bmatrix} r \\ \theta \\ \dot{r} \\ \dot{\theta} \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ 0 \\ 0.345 \end{bmatrix} T + \begin{bmatrix} 0 \\ 0 \\ 0 \\ 0.345 \end{bmatrix} d$$

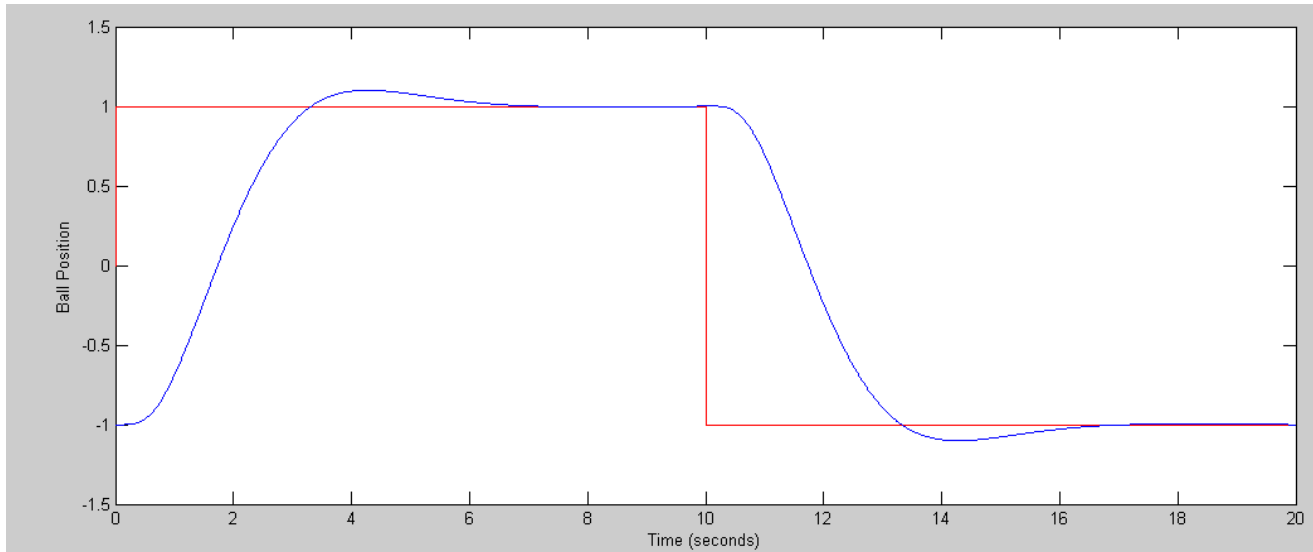
Full-State Feedback with Constant Disturbances

1) For the nonlinear simulation, use the feedback control law you computed in homework #6

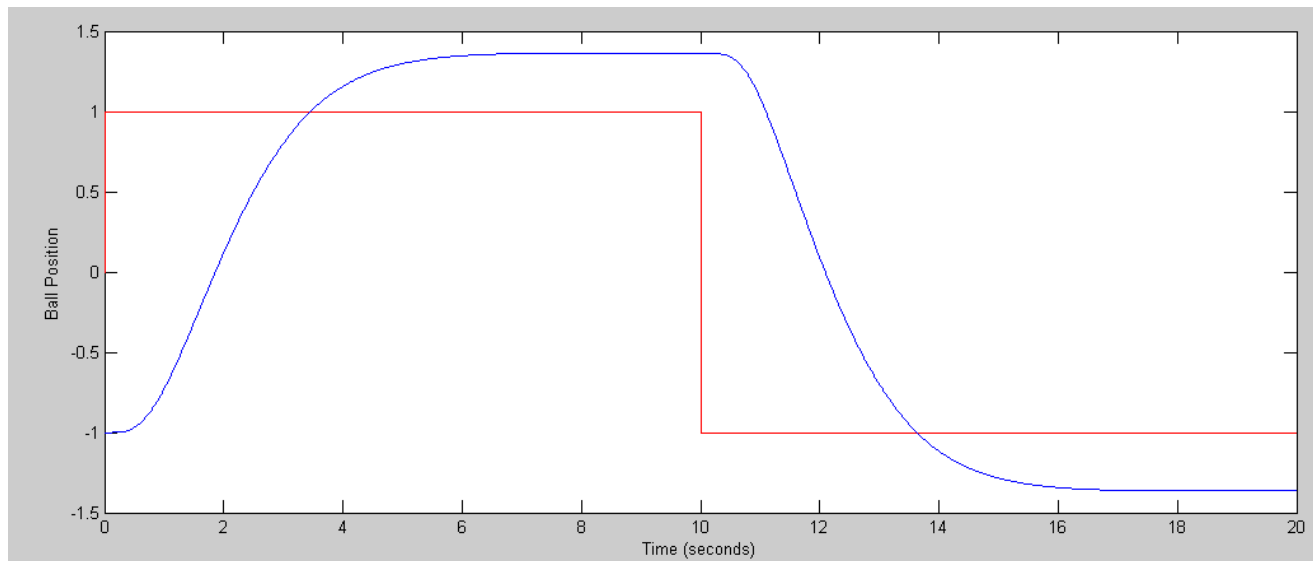
- With $R = 1$ and the mass of the ball = 2.2kg (same result you got for homework #6), and
- With $R = 1$ and the mass of the ball decreased to 2.5kg

(i.e. a constant disturbance on the system due to a different mass of the ball)

$$K_x = \begin{bmatrix} -32.6762 & 103.6051 & -18.2583 & 30.7246 \\ -11.1284 & 0 & 0 & 0 \end{bmatrix}$$

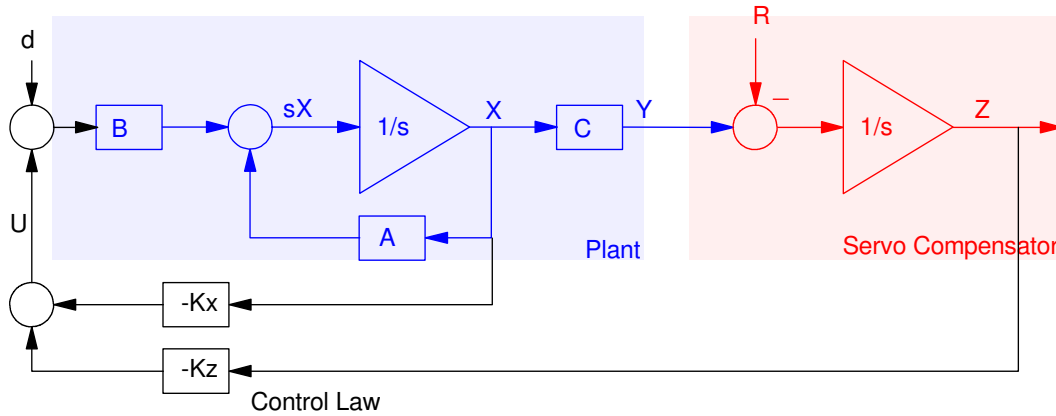


Step Response when $m = 2.2\text{kg}$
nonlinear simulation



Step Response when $m = 2.5\text{kg}$
nonlinear simulation

Servo Compensators with Constant Set-Points



2) Assume a constant disturbance and/or a constant set point. Design a feedback control law that results in

- The ability to track a constant set point ($R = \text{constant}$)
- The ability to reject a constant disturbance ($d = \text{constant}$),
- A 2% settling time of 6 seconds, and
- No overshoot for a step input.

```
>> A5 = [A, zeros(4,1) ; C, 0]
```

```

    0         0         1.0000         0         0
    0         0         0         1.0000         0
    0        -7.0000         0         0         0
   -7.4370         0         0         0         0
    1.0000         0         0         0         0

```

```
>> B5u = [B ; 0]
```

```

    0
    0
    0
    0.3450
    0

```

```
>> B5r = [0*B ; -1]
```

```

    0
    0
    0
    0
   -1

```

```
>> C5 = [1,0,0,0,0];
```

```
>> D5 = 0;
```

```
>> K5 = ppl(A5, B5u, [-4/6, -4/6, -3, -4, -5])
```

```
K5 =  -63.3325  183.8969  -53.0021  38.6473  -11.0421
```

```
>> Kx = K5(1:4)
```

```
Kx =  -63.3325  183.8969  -53.0021  38.6473
```

```
>> Kz = K5(5)
```

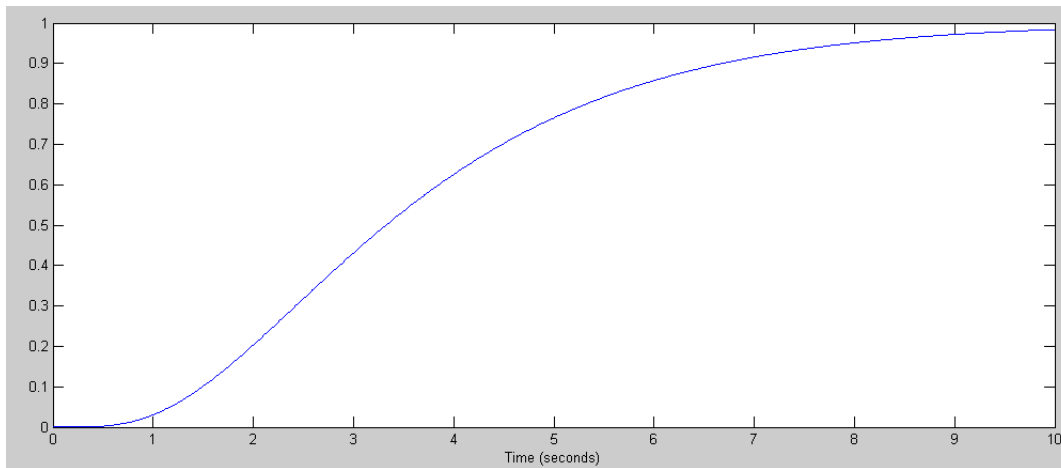
```
Kz =  -11.0421
```

3) For the linear system, plot the step response

- With respect to a step change in R, and
- With respect to a step change in d

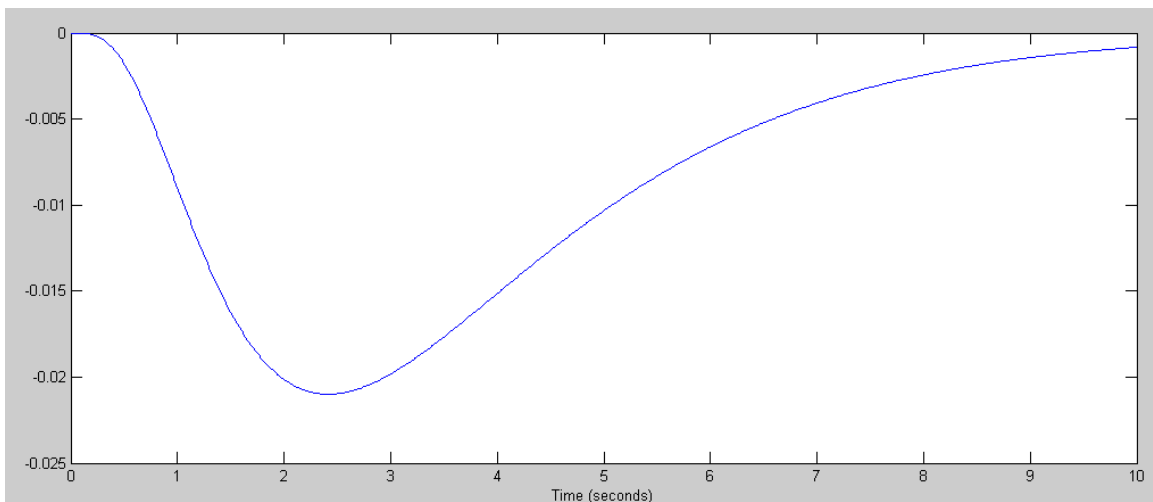
Form the closed-loop system

```
>> G5 = ss(A5 - B5u*K5, B5r, C5, D5);  
>> t = [0:0.01:10]';  
>> y = step(G5,t);  
>> plot(t,y);  
>> xlabel('Time (seconds)');  
>>
```



Response to a step input in Ref

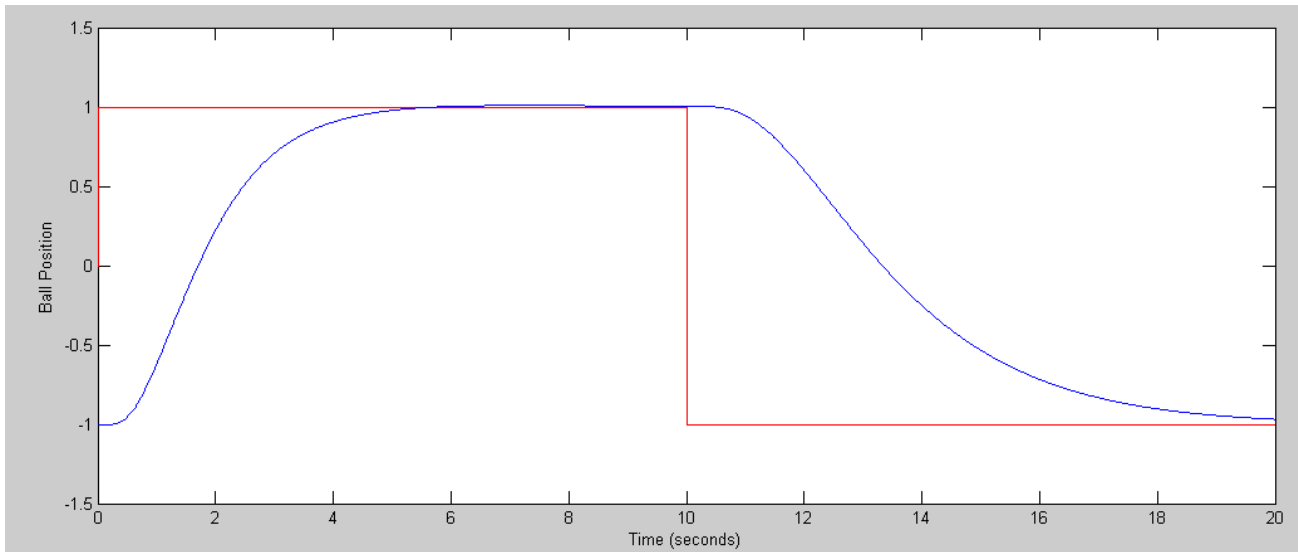
```
>> B5d = [B ; 0];  
>> G5 = ss(A5 - B5u*K5, B5d, C5, D5);  
>> y = step(G5,t);  
>> plot(t,y);  
>> xlabel('Time (seconds)');
```



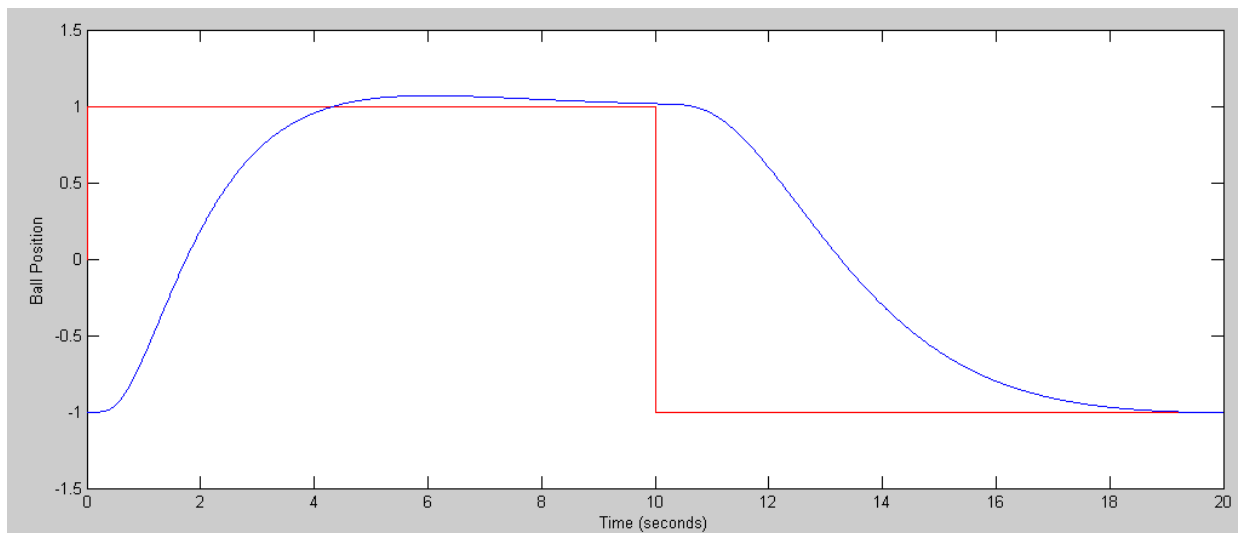
Response in a step change in the disturbance

4) Implement your control law on the nonlinear ball and beam system

- With $R = 1$ and the mass of the ball being 2.2kg, and
- With $R = 1$ and the mass of the ball being 2.5kg



Step response with $m = 2.2\text{kg}$



Step Response when $m = 2.5\text{kg}$

Code:

```
% Ball & Beam System

X = [-1, 0, 0, 0]';
dt = 0.01;
t = 0;

% Servo Compensator
Kx = [-63.3325 183.8969 -53.0021 38.6473];
Kz = -11.0421;
Z = 0;

n = 0;
y = [];

while(t < 20)
    Ref = sign(sin(t*pi/10));
    U = -Kx*X - Kz*Z;

    dX = BeamDynamics(X, U);
    dZ = X(1) - Ref;

    X = X + dX * dt;
    Z = Z + dZ * dt;

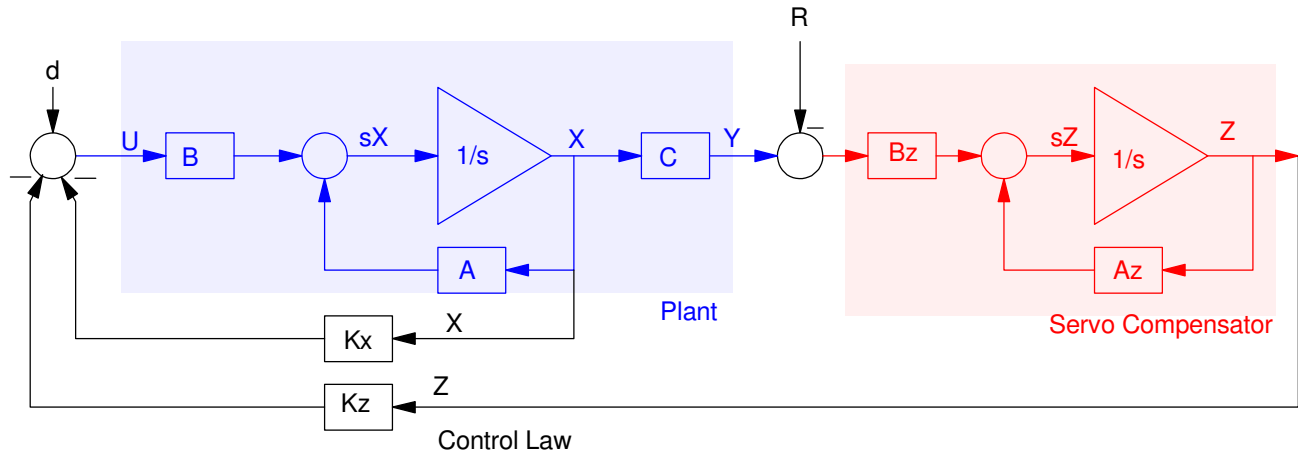
    t = t + dt;

    y = [y ; Ref, X(1)];
    n = mod(n+1,5);
    if(n == 0)
        BeamDisplay(X, Ref);
    end
end

t = [1:length(y)]' * dt;

plot(t,y(:,1),'r',t,y(:,2),'b');
xlabel('Time (seconds)');
ylabel('Ball Position');
```

Servo Compensators with Sinusoidal Set-Points



5) Assume a 0.6 rad/sec disturbance and/or set point (R). Design a feedback control law that results in

- The ability to track a constant set point ($R = \sin(0.6t)$)
- The ability to reject a constant disturbance ($d = \sin(0.6t)$),
- A 2% settling time of 12 seconds, and

First, define a servo compensator with poles at $\pm j0.6$

```
>> Az = [0, 0.6; -0.6, 0]
```

```
      0      0.6000
 -0.6000      0
```

```
>> eig(Az)
```

```
      0 + 0.6000i
      0 - 0.6000i
```

```
>> Bz = [1; 1];
```

Next, form the augmented system

```
>> A6 = [A, zeros(4,2) ; Bz*C, Az]
```

```
      0      0      1.0000      0      0      0
      0      0      0      1.0000      0      0
      0     -7.0000      0      0      0      0
 -7.4370      0      0      0      0      0
  1.0000      0      0      0      0      0.6000
  1.0000      0      0      0     -0.6000      0
```

```
>> B6u = [B ; 0; 0]
```

```
      0
      0
      0
  0.3450
      0
      0
```

```
>> B6r = [0*B ; -Bz]
```

```
0
0
0
0
-1
-1
```

```
>> C6 = [C, 0, 0];
```

```
>> D6 = 0;
```

Find the feedback gains using pole-placement. Since the open-loop poles are at {2.68, -2.68, 2.68i, -2.68i}, place them here (somewhat arbitrary):

```
>> eig(A)
```

```
-2.6861
0.0000 + 2.6861i
0.0000 - 2.6861i
2.6861
```

```
>> K6 = ppl(A6, B6u, [-0.5, -0.5, -0.5+3i, -0.5-3i, -2.7, -2.8])
```

```
K6 = -72.7410 83.1884 -32.9896 21.7391 -2.2420 -20.8874
```

```
>> Kx = K6(1:4)
```

```
Kx = -72.7410 83.1884 -32.9896 21.7391
```

```
>> Kz = K6(5:6)
```

```
Kz = -2.2420 -20.8874
```

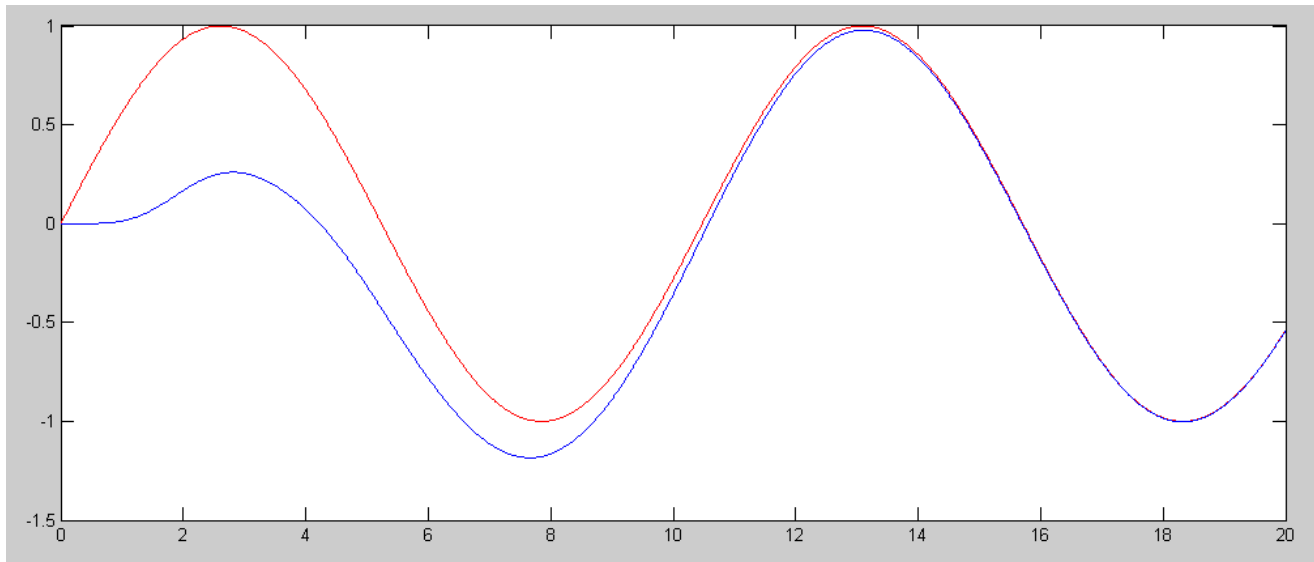
```
>>
```


6) For the linear system, plot the response

- With $R(t) = \sin(0.6t)$, and
- With $d(t) = \sin(0.6t)$

Response to $R(t)$

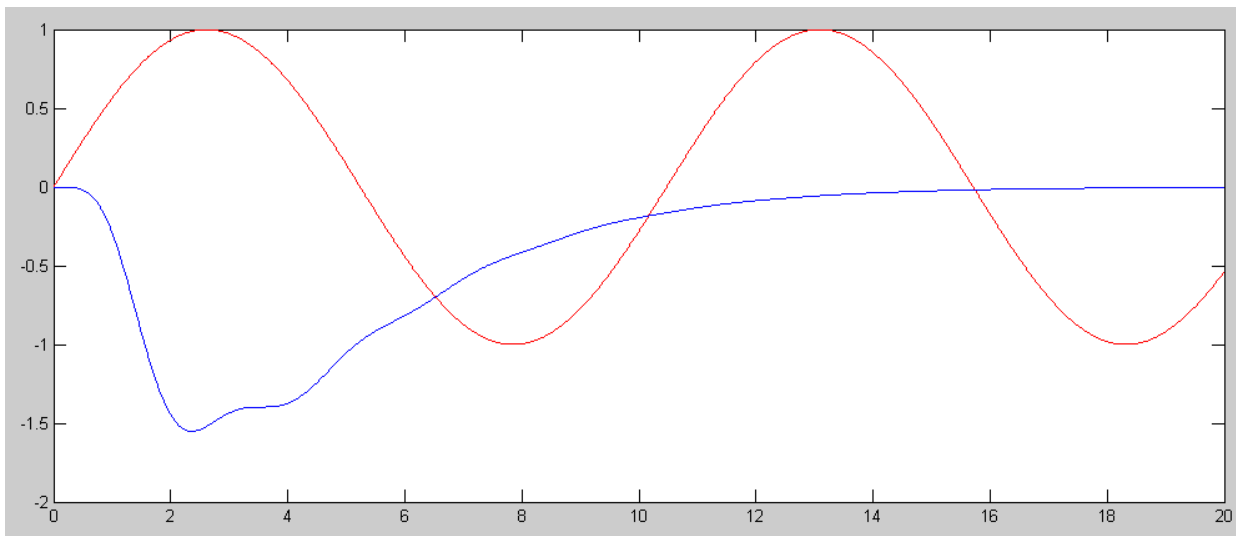
```
>> G6 = ss(A6 - B6u*K6, B6r, C6, D6);  
>> t = [0:0.01:20]';  
>> R = sin(0.6*t);  
>> X0 = zeros(6,1);  
>> y = step3(A6-B6u*K6, B6r, C6, D6, t, X0, R);  
>> plot(t,R,'r',t,y,'b')
```



Response to $R(t) = \sin(0.6t)$. red = $R(t)$, blue = $y(t)$

Response to $d(t)$

```
>> B6d = [B ; 0*Bz];  
>> y = step3(A6-B6u*K6, B6d, C6, D6, t, X0, R);  
>> plot(t,R,'r',t,y*100,'b')
```

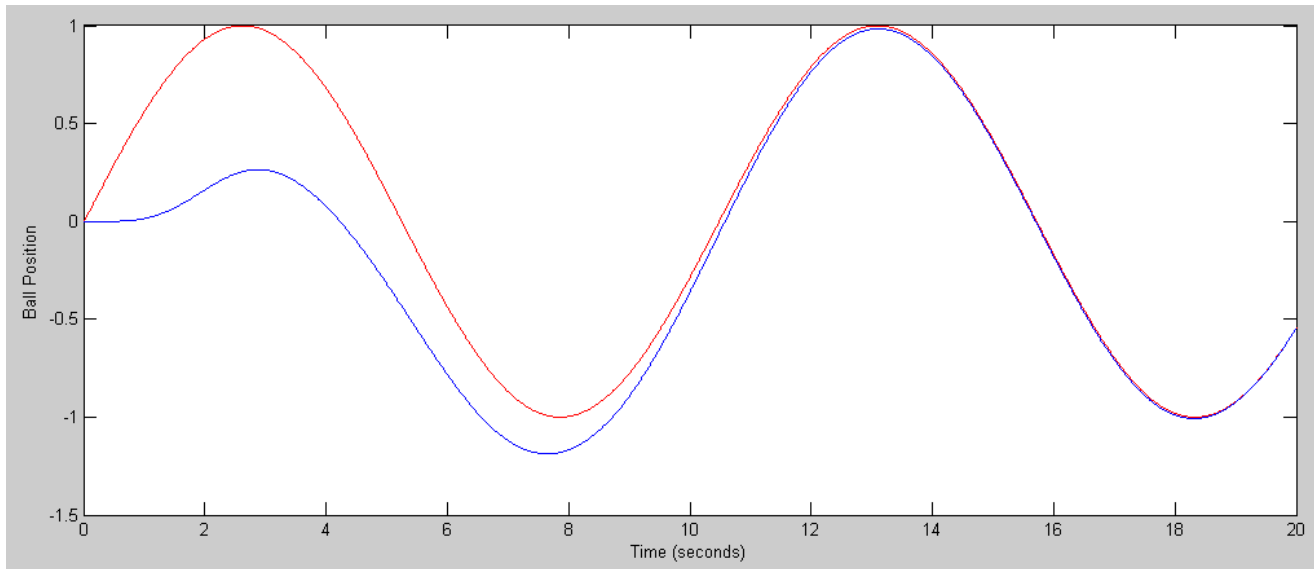


Response to $d(t) = \sin(0.6t)$. red = $d(t)$, blue = $100*y(t)$

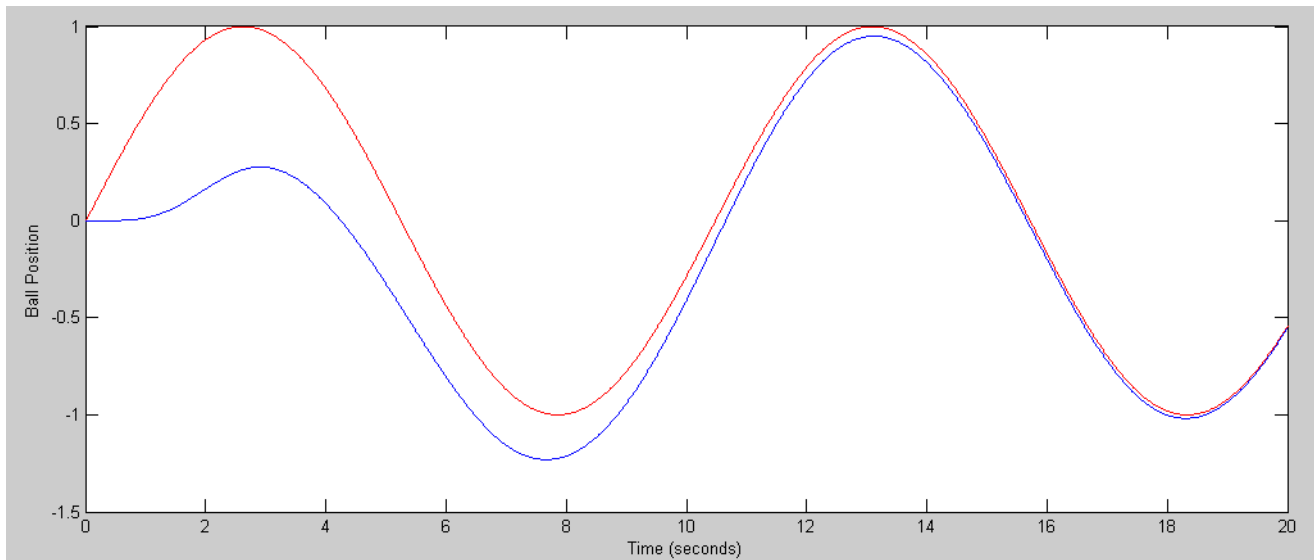
7) Implement your control law on the nonlinear ball and beam system

- With $R = \sin(0.6t)$ and the mass of the ball being 2.2kg, and
- With $R = \sin(0.6t)$ and the mass of the ball being 2.5kg

Nominal Case: $m = 2.2\text{kg}$



$m = 2.5\text{kg}$



Code:

```
% Ball & Beam System

X = [0, 0, 0, 0]';
dt = 0.01;
t = 0;

% Servo Compensator at 0.6 rad/sec
Az = [0, 0.6 ; -0.6, 0];
Bz = [1; 1];
Kx = [-72.7410  83.1884  -32.9896  21.7391];
Kz = [-2.2420  -20.8874];
Z = zeros(2, 1);

n = 0;
y = [];

while(t < 20)
    Ref = sin(0.6*t);
    U = -Kx*X - Kz*Z;

    dX = BeamDynamics(X, U);
    dZ = Az*Z + Bz*(X(1) - Ref);

    X = X + dX * dt;
    Z = Z + dZ * dt;

    t = t + dt;

    y = [y ; Ref, X(1)];
    n = mod(n+1, 5);
    if(n == 0)
        BeamDisplay(X, Ref);
    end
end

t = [1:length(y)]' * dt;

plot(t, y(:, 1), 'r', t, y(:, 2), 'b');
xlabel('Time (seconds)');
ylabel('Ball Position');
```